Nonlinear Vibrations of Aerospace Structures

University of Liège, Belgium



Construction of an accurate mathematical model of the dynamics of a real structure based on input and output measurements.

System Identification in Structural Dynamics?

Construction of an accurate mathematical model of the dynamics of a real structure based on input and output measurements.



NSI in the Design Cycle of Engineering Structures



Why is nonlinear system identification difficult?

Driving factors of progress over the past decade.

NSI: a three-step process.

Two good excitation signals.

Nonlinearity detection in sine and broadband conditions.

Challenge #1: Sensitivity to the type of input signal.

Distorted resonance under sine excitation.

Distorted resonance under random excitation.





Challenge #2: Specific and demanding test campaigns.

- Increase the sampling frequency.
- Record throughput time histories.
- Measure the force signal.
- Instrument potential nonlinearities with sensors on both sides.
- Consider multiple levels of excitations/types of excitations.
- Study multiple sets of initial conditions, and reverse the sweep.

Challenge #3: Inapplicability of linear concepts.

Modal properties are not invariant.

FRFs are not invariant.

Frequency (Hz) Amplitude (dB) Amplitude (dB) Frequency (Hz) Energy (J) Frequency (Hz)

Challenge #4: Individualistic nature of structural nonlinearities.



Challenge #5: Limited amount of prior knowledge.

$$\mathbf{M} \, \ddot{\mathbf{q}} + \mathbf{C}_{v} \, \dot{\mathbf{q}} + \mathbf{K} \, \mathbf{q} + \mathbf{g}(\mathbf{q}, \dot{\mathbf{q}}) = \mathbf{p}$$

Determining nonlinear functional forms is an integral part of nonlinear system identification (and arguably the most difficult).

Challenge #6: True system may be outside the model class.



Where Do We Stand?

First analyses of SDOF systems.

MDOF systems and continuous structures with localised nonlinearities. Real-life structures with strong nonlinearities, nonlinear modal analysis, numerical model updating.



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#1: Real Structures Are Nonlinear...



tail-cone

Landing gears

Wing-engine pylons, engine mounts

New materials (*e.g.*, composites)

...and (Linear) Commercial Software Fail to Identify Them!



#2: Environmental Constraints Boost Technological Progress



In 2050 technologies and procedures available allow a 75% reduction in CO2 emissions per passenger kilometre [...] and a 90% reduction in NOx emissions. The perceived noise emission [...] is reduced by 65%.

#3: Increased Maturity of Connected Fields



Recent Progress in the Theory of Nonlinear Vibrations

Bifurcation theory and nonlinear normal modes (NNMs).



Single-aisle aircraft bifurcation diagram [B. Krauskopf, 2015].

Spectral submanifold of a 2-DOF system [G. Haller, 2016].

Recent Progress in the Computation of Nonlinear Responses

Tailored shooting, harmonic balance and collocation algorithms.



Phase diagram of a vibro-impact system [B. Cochelin, 2014].

NNM of a compressor blade with friction [F. Thouverez, 2009].

Basins of attraction and experimental continuation.



Experimental impact oscillator [J.J. Thomsen, 2014].

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Importance of the Toolbox Philosophy in NSI



Do I observe nonlinear effects? Yes. Should I build a nonlinear model? Yes.

Where is the nonlinearity located? At the joint. What is the underlying physics? Dry friction. How to model its effects? $f_{nl}(q, \dot{q}) = c \, sign(\dot{q})$.

Model parameters? c = 5.47.

How uncertain are they? $c = \mathcal{N}(5.47, 1)$.

Evidence nonlinear behaviour in experimental data.

This is not particularly challenging:

A wide variety of techniques exists in the literature.Harmonic and random excitations can be addressed.A single excitation level is generally sufficient.

This is crucial:

Detection supports an important decision as building a nonlinear model demands more time, capabilities and efforts than a linear model.

Infer a suitable nonlinearity model from experimental data.

This is challenging:

Prior knowledge is most often very limited.

Physical mechanisms resulting in nonlinearity are extremely diverse.

Nonlinearity may translate into a plethora of dynamic phenomena.

This is crucial:

The success of the parameter estimation step is conditional upon an accurate characterisation of all observed nonlinearities.

Fit a mathematical model to experimental data.

This is challenging:

If characterisation was not successfully completed (black-box models).

If nonlinearities were not instrumented.

If the input signals were not measured (operational NSI).

This is crucial:

Quantitative models are required to carry out response prediction, numerical model updating and validation, design optimisation, ...

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Two "Good" Input Signals: Sine Sweep and Random Multisine

Linear sine sweep:

$$u(t) = A \sin\left(2\pi f_0(t-t_0) + 2\pi \frac{r}{2}(t-t_0)^2 + \varphi_0\right)$$



Forcing frequency (Hz)



Linear sine sweep:

$$u(t) = A \sin\left(2\pi f_0(t-t_0) + 2\pi \frac{r}{2}(t-t_0)^2 + \varphi_0\right)$$

Fully deterministic signal, so easy to interpret data visually. Strongly activates nonlinearities, as energy is concentrated. Very useful in nonlinearity characterisation.

Logarithmic sweep may be used to focus on low frequencies.

Two "Good" Input Signals: Sine Sweep and Random Multisine

Log sine sweep:

$$u(t) = A \sin\left(\frac{2\pi f_0}{\log(2) r} \left(2^{r(t-t_0)} - 1\right) + \varphi_0\right)$$



Forcing frequency (Hz)





Multisine with uniformly-distributed random phases:

$$u(t) = N^{-1/2} \sum_{k} U_{k} \exp\left(j2\pi k \frac{f_{s}}{N} t + j\varphi_{k}\right)$$
Amplitude (dB)

Frequency (Hz)

$$u(t) = N^{-1/2} \sum_{k} U_{k} \exp\left(j2\pi k \frac{f_{s}}{N} t + j\varphi_{k}\right)$$

Phase (rad)

$$3.14$$

$$-3.14$$

Frequency (Hz)

Frequency (Hz)

Multisine with uniformly-distributed random phases:

$$u(t) = N^{-1/2} \sum_{k} U_k \exp\left(j2\pi k \frac{f_s}{N} t + j \varphi_k\right)$$

Random appearance in TD, but deterministic amplitudes in FD.

All frequencies excited throughout the test.

Periodicity allows to remove transients and characterise noise.

Gaussian signal for a sufficiently large number of frequencies.

Very useful in nonlinear parameter estimation.

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A First Test Case: the SmallSat Spacecraft from Airbus DS





Challenges:

- Nonsmooth nonlinearities
- High damping
- Gravity-induced asymmetry
- Viscoelastic components

WEMS Device: Piecewise-linear Behaviour



Displacement

Isolation and protection device.

Experimental stiffness curve.

Envelope-based Analysis of a Raw Time History



Skewness and Nonsmoothness Result in a Jump



Discontinuity in Slope due to a Clearance in the System



Asymmetry owing to Gravity-induced Prestress



Breakdown of the Principle of Superposition



Sweep frequency (Hz)

Existence of Nonlinear Hysteresis



Transient Effect in Sweep-Up and Sweep-Down Testing

Relative displacement (–)



A Close-up Reveals the Appearance of Strong Harmonics

Relative displacement (-)



A Second Test Case: an F-16 Fighter Aircraft



F16 aircraft, Saffraanberg, Belgium.



Joints between substructures are generic sources of NLs.

Force Spectrum May Exhibit Drop-off at Resonance



The Input Spectrum Is Accurately Realised



Digression: Force Realisation in Sine-Sweep Testing



Transient Decay Over Multiple Periods of Measurement



Superposed FRFs do not Match at Two Excitation Levels

Amplitude (dB)



Digression: Frequency Mixing in Nonlinear Systems

$$u(t) = 2\cos(\omega t) = e^{j\omega t} - e^{-j\omega t}, \omega = 1$$

$$\underbrace{\bullet}_{-3 -2 -1} \underbrace{\bullet}_{0 -1 -2 -3} \underbrace$$

Output of a cubic nonlinearity:

$$y(t) = u^{3}(t)$$

= $(e^{j\omega t} - e^{-j\omega t})(e^{j\omega t} - e^{-j\omega t}) (e^{j\omega t} - e^{-j\omega t})$

Output of a cubic nonlinearity:

$$y(t) = u^{3}(t)$$

= $(e^{j\omega t} - e^{-j\omega t})(e^{j\omega t} - e^{-j\omega t}) (e^{j\omega t} - e^{-j\omega t})$

All possible combinations, 3 by 3, of the frequencies -1 and 1.



NL Detection using a Carefully-Selected Input Spectrum



F-16 Measurement at Low Excitation Level

Amplitude (dB)



Odd and Even NLs Are Clearly Detected at High Level

Amplitude (dB)



The Source of Nonlinearity ... See Next Lecture



The Source of Nonlinearity ... See Next Lecture



Nonlinear system identification is a three-step process.

Nonlinearity detection is the most straightforward step, but supports an important decision in the design cycle.

Sine and broadband excitations are treated separately (toolbox).

Next lecture is dedicated to characterising nonlinearity.

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T. Dossogne, J.P. Noël, C. Grappasonni, G. Kerschen, B. Peeters, J. Debille, M. Vaes, J. Schoukens, **Nonlinear ground vibration identification of an F-16 aircraft – Part II**, Proceedings of IFASD, Saint Petersburg, Russia, 2015.