Nonlinear Vibrations of Aerospace Structures

Tutorial 01

Introduction to NI2D



T01 Nonlinear Vibrations Course at ULiège

Check www.nolisys.com

You'll get free access to the NI2D software.

Matlab environment; stand alone .exe.





NI2D Philosophy

Nonlinear Identification to Design

- A prototype of the structure is available:
- \rightarrow Test it, identify the nonlinearities and upgrade the linear FEM.
- A priori knowledge about the nonlinearities is available:
- \rightarrow Load the linear FEM into NI2D and implement the nonlinearities using NI2D elements library.

How To Read the Slides







Launch NI2D and activate your license under *Preferences* ...

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Create a New Model

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The Different Solvers





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	mouse-right control+mouse-left shift+mouse-left	Menus on tab background and model elements Add element between masses Enable/disable elements		
	alt+mouse-left double+mouse-left f2	Select/unselect masses Edit element or mass Send graphic to Windows clipboard		
	f3 f4 f5	Send graphic to image file (automatic choice) Clone current NI2D window Run selected solver		
_	f7 f8 f12	Add to report (automatic choice) Add to report (editing title/comments) Result manager		
	escape tab	Reset window Last two results displayed		
	% control+c	NI2D command prompt Copy curve in NI2D clipboard Reset 2D zoom		
	0 U	Reset initial conditions User colors for selected masses		
The help menu is ada	apted to	each tab		
Model	control+v Identify Simulate	Select and paste settings (model, views, parameters) Understand Design User		

Launch Your First Numerical Simulation

Consider a Thin Short Beam ...

... connected to a cantilever beam (ECL benchmark)



Linear model identified at low level (31 Hz, 0.12%):

 $0.289\ddot{x} + 0.1357\dot{x} + 11009x = F\sin\omega t$

Create a New Model: 1 DOF Linear Oscillator

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	Mass:	0.289	Kg
	Linear damping:	0.1357	N.s/m
	Linear stiffness:	11009	N/m
Continu	e >	Abort	

You Can Change the Coefficients Anytime ...



Add an External Force

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The Final Linear Model

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Calculate the Time Response with Newmark





How to Choose the Right Sampling Frequency?

				Accur	acy
			Stability limit	Amplitude error	Periodicity error
Algorithm	γ	β	ωh	$\rho - 1$	$\frac{\Delta T}{T}$
Purely explicit	0	0	0	$\frac{\omega^2 h^2}{4}$	_
Central difference	$\frac{1}{2}$	0	2	0	$-\frac{\omega^2 h^2}{24}$
Fox & Goodwin	$\frac{1}{2}$	$\frac{1}{12}$	2.45	0	$O(h^3)$
Linear acceleration	$\frac{1}{2}$	$\frac{1}{6}$	3.46	0	$\frac{\omega^2 h^2}{24}$
Average constant acceleration	$\frac{1}{2}$	$\frac{1}{4}$	œ	0	$\frac{\omega^2 h^2}{12}$
Average constant acceleration (modified)	$\frac{1}{2} + \alpha$	$\frac{(1+\alpha)^2}{4}$	œ	$\alpha - \frac{\omega^2 h^2}{2}$	$\frac{\omega^2 h^2}{12}$

Now: Run



Save Your Results in a Curve Stack



Check Your Results Anytime



Frequency Response Functions

The Linear Model



Calculate the Linear FRF ...



And Check the Result



Compare the Time and the Frequency Response



You Can Tag Your Results (F11) ...



And Manage All Your Results (F12)



Launch Your First Nonlinear Simulation

Nonlinear Model of the 1st Beam Mode



Linear model identified at low level (31 Hz, 0.12%):

 $0.289\ddot{x} + 0.1357\dot{x} + 11009x = F\sin\omega t$

Nonlinearity identified at high level: $2.37 \cdot 10^9 x^3$

Upgrade the Linear Model from T02 ...



With a Cubic Spring



Run the Newmark Time Integration Again ...



And Save Your Results For Comparison



You Can Scroll through Previous Results



And Save Different Results in the Same Curve Stack



Rename and Arrange Everything in Your Graph ...



What can we observe?

Nonlinearity Introduces a Fundamental Change



The Nonlinear System Response Has a Greater Bandwidth ...



The Resonance Frequency Shifts



A Jump Downwards Can Be Observed



Let's Reverse the Sweep

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We Can Jump Up Too!



Calculate the Nonlinear FRF



But Set Appropriate Numerical Parameters First

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					Starting point:		30	Hz			<u> </u>
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		V V V		Optimal num	ber of iterations:		3				
				Maximum n	umber of points:		10000				
					Beta angle:		90	0			
Mod	el Ident	ify Simulat	НВ р	arameters	Appl	у	Start		Cancel		

But Set Appropriate Numerical Parameters First

File Model Identify Simulate Understand Design User Help	
N Harmonic Balance parameters - X -	
Number of harmonics:	%
Number of points:	
Compute stability Reordering	
Linear mode: Mode #1 (31.063125 Hz) ~	
Amplitude of 1st guess: 0.001 m	
Maximum number of iterations: 15	
Relative precision: 1e-06	
Scaling factor for displacements: 1e-06	
Scaling factor for time: 1	
Apply Cancel	
Model Identify Simulate Understand Design User	

And Change the Excitation Signal to a Sine

File Model Ider	fy Simulate Understand Design User Help ↓ 🗄 🕨 - 🖾 💡 😵 🛛 🗍 🗳 ▲ 🗢 🟹 🗍	Ⅱ ■
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	Sine Sweep Random User Measure	~
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The Nonlinear FRF



The Jump Phenomenon Can Be Understood



The Jump Phenomenon Can Be Understood



The Jump Phenomenon in Practice

(See the attached video)



The video shows the dynamics of a clamped-clamped thin beam with geometrical nonlinearities during a sweep over the second beam mode.

Compare to the Linear Frequency Response



Sweep with Different Forcing Amplitudes





Sine with a Fixed Frequency in the Multi-valued Region

Now: Consider an Excitation with a Fixed Frequency



Run the Newmark Integration and Save



Change the Initial Conditions



Run and Save in the Same Curve Stack



Compare



The Nonlinear FRF Helps to Understand

