Nonlinear Vibrations of Aerospace Structures

Tutorial 06 Nonlinear Simulations

- Nonlinear Modelling
- Time Integration
- Continuation



Get familiar with NI2D tools for nonlinear simulations:

Simulate the system dynamics.

Compute nonlinear frequency response curves (NFRCs).

Case Study: A Nonlinear Beam

Linear FE model



Identified Nonlinearities



Cubic coefficient (geometrical)

Quadratic coefficient (clamping)

Import the Linear FE Model

1

▶ NI2D: New model
User models Spring/mass system MCK matrices Finite element model DAQ model Measured signals Hz Mass:
Stiffness:
, Vx

- Create a new model (1) and select the matrices M,C and K (2).
- Import **M**, **C** and **K**.
- Name the new system 'NLBeam'.

At this point, the model is linear and a nonlinear connection (cubic + quadratic) has to be created between the displacement at the tip of the main beam (DOF #28) and the ground.



Create Nonlinear Connections

1. Create 'Nonlinear polynomial stiffness'

N Add element on connection	—		×			
Linear stiffness						
Nonlinear polynomial stiffness						
Nonlinear piecewise linear stiffness						
Nonlinear contact Nonlinear polynomial damping						
Coulomb friction Trilinear damping						
Point-by-point damping						
Hysteretic damping (Bouc-wen)						
			×			
OK Cancel						

2. Create an odd cubic nonlinearity with stiffness set to 8×10^9 N/m³



Create Nonlinear Connections

- Create a new connection between DOF #28 and the ground
- Create an even quadratic nonlinearity with stiffness set to $-1.05 \times 10^7 \text{ N/m}^2$

▶ Polynomial NL spring n ×						
Coefficient:	-1.05e7 N/m^x					
Exponent:	2					
O odd even O unilateral						
Apply	Cancel					

Visualise Nonlinear Connections

 Use 'right click' in the nonlinear connections and select 'View' to visualise each restoring force.





As a first exercise, we will study the system's response to a sine excitation with a frequency close to the first resonance frequency of the beam (31.28 Hz).



Create a Sine External Force

 Create a forcing by using 'right click' DOF #8 and selecting 'Add external force'.



Create a Sine External Force

• Select 'Sine' and give the following forcing parameters, then 'Apply'.

NE	xternal force	on dof n°{	3			—		\times
Sine	Sine Sweep	Random	User	Measure				
			Amp Frequ F	litude: Jency: Phase:	3 34 þ		N Hz °	
	Apply			Newmark (F5)			Cancel	

 In order to show the time series at the tip of the main beam, use 'Alt + double left click' on DOF #28 to display next results on that DOF.



You can modify the colour associated to DOF #28 using 'right click' on that DOF.

• Go to the 'Simulate' tab (1) and select 'Newmark' as a solver (2).



• Set the parameters for the solver and click on 'Apply' for both windows.

File Model Identify Simulate Understand	Design User Help		
	Newmark parameters [SINE]		
1	Final time:	20	sec
	Time step:	0.01	sec
	Number of time steps:	2000	2
	Number of periods:	100	
	Time steps by period:	90	
	Saved dofs:) all	⊖ selected	
Degree of freedom all \checkmark \Rightarrow 0 3	Initial cond. Apply	Run (F5)	Cancel
Displacement: 0 m			
Velocity: 0 m/sec			
Apply Cancel]		

• Tag your result 'dt = 0.01s' using 'F11' and save it in a new curve stack.



- Perform another simulation for a smaller time step of 0.001s, tag it, and select a different colour using 'right click' on the curve. Compare the curves by saving them in the same curve stack.
- Repeat the same operations for smaller time steps of 0.0002s and 0.0001s.

Influence of the Time Step

 Go to the 'User' tab, and use 'right click' and 'Stack up/down' or 'Auto stack' to observe the different curves in the figure.



Question:

Can you explain why the time series are different? Question:

Can you explain why the time series are different?

Answer:

Selecting a small time step is crucial for obtaining accurate time series. For example, linear Newmark's scheme has a periodicity error of

$$\Delta \omega = \frac{4 \pi^2 f^2}{12 f_s^2}$$
Frequency of interest
in the signal
Sampling frequency

For an error of less than 1%, and for f = 34 Hz, one should have larger f_s than 620 Hz.

Considering that third harmonics are present in the response, f_s should be even larger. Here we can choose 0.0002s as an optimal time step.



Influence of the Initial Conditions

- Select a time step of 0.002s, and consider an initially perturbed system using an initial velocity of 1 m/s for DOF #28.
- By using tags and colours, compare the initially perturbed and unperturbed responses.

Depending on the initial conditions, the system can have small or large amplitude oscillations.

Bistable behaviour





We consider the system from exercise 1 again:

As a second exercise, we will study the system's response to a **swept-sine excitation** with a frequency range between 25 Hz and 40 Hz, which encompasses the first resonance frequency of the beam (31.28 Hz).

We will study the effect of the sweep rate, and compare responses to sweep up and down in order to highlight the bistable region in the frequency response.



Create a Swept-sine External Force

• Select 'Sine Sweep' on DOF #8 and give the following forcing parameters. Apply.



 In order to show the time series at the tip of the main beam, use 'Alt + left click' in DOF #28 to display next results on that DOF.



• In the 'Simulate' tab, modify the solver parameters to set all displacements and velocities to zero, and the time step to 0.0002s. Apply.

Final time:	90	sec	
Time step:	0.0002	sec	2
Number of time steps:	450000		N Initial conditions -
Number of periods:			Degree of freedom all ∨ →0
Time steps by period:			Displacement: 0
			Velocity: 0

Model Identify Simulate Jnderstand Design User

• Start the simulation and tag your result as 'v = 10 Hz/min' by using 'F11'. Displacement vs time, dof n°28



The response is asymmetric due to the presence of the quadratic nonlinearity. The jump down occurs due to hardening behaviour.

• Reverse the sweep direction.



Comparison between Sweep Up and Sweep Down

• Start the simulations and tag the result as 'v = 10 Hz/min D'.



The jump up occurs due to hardening behaviour.

Comparison between Sweep Up and Sweep Down

• Compare the responses for a sweep up and sweep down.



The bistable region spans between 32 Hz and 37 Hz.

Go back to your model. Select a sine excitation with the following parameters.

N E	xternal force	on dof n°8	3		_		×	
Sine	Sine Sweep	Random	User	Measu	re			
			Amp	litude:	3		Ν	
			Frequ	iency:	34		Hz	
			F	hase:	0		۰	
	Apply			Newma	rk (F5)		Cancel	

The chosen forcing amplitude of 3 N will be considered for all the periodic solutions along the branch.

• Modify the continuation parameters:



• Modify the HB parameters:

Number of harmonics N_H retained in the Fourier series N Harmonic Balance parameters _ Number of harmonics: F 5 Number of points: 512 Number of the samples N in Ъ the Fourier transform. Compute stability Reordering Linear mode: Use of symmetric reverse Cuthill-McKee permutation Amplitude of 1st guess: 0.001 m to accelerate the eigenvalue Maximum number of iterations: 15 problem resolution. Relative precision: 1e-06 Scaling factor for displacements: 5e-06 auto Scaling factor for time: 3000 Apply Cancel

• Modify the HB parameters:

N Harmonic Balance parameters		_		×	
Number of harmonics:		▶ 5			
Number of points:		▶ 51	2		
Compute stability	Reordering				Amplitude of the sine series
Linear mode:					used as an initial guess for all DOFs.
Amplitude of 1st guess:	0.001	m	γ		
Maximum number of iterations:	15				The Newton-Raphson
Relative precision:	1e-06				of iteration is exceeded.
Scaling factor for displacements:	5e-06				
Scaling factor for time:	3000] auto		The Newton-Raphson procedure stops if the relative
Apply		Cancel			error is smaller than this precision.

• Modify the HB parameters:

N Harmonic Balance parameters	- 🗆 X
Number of harmonics:	▶ 5
Number of points:	▶ 512
Compute stability	dering
Linear mode:	
Amplitude of 1st guess: 0.).001 m
Maximum number of iterations:	15
Relative precision: 16	le-06
Scaling factor for displacements: 56	ie-06
Scaling factor for time: 3	3000
Apply	Cancel

Because the frequency (here, around 30 Hz = 188 rad/s) and the amplitude (here, around 0.001 m)
have different order of magnitude, time and displacements have to be rescaled to avoid ill conditioning.

 Start the continuation procedure and wait until the maximum frequency (40 Hz) is reached.



 Start the continuation procedure and wait until the maximum frequency (40 Hz) is reached.



 Use 'double left click' on a point to represent its time series reconstructed from the Fourier coefficients, and its Floquet exponent/multipliers.



• You can choose more results to display.



 Use 'double left click' on a point to represent its time series reconstructed from the Fourier coefficients, and its Floquet exponent/multipliers.



• Go back to the NFRC with a 'right click'.



 Among the results to display, select the evolution of the harmonic components along the curve.



 Among the results to display, select the evolution of the harmonic components along the curve.



 Add the nonlinear frequency response in the stack curves of the swept-sine responses.



A small step size is necessary for accurate time integrations
(sampling frequency should be approximately 200 times higher than the frequency of interest).

A small sweep rate is necessary to accurately represent amplitude jumps up and down.

Sine and swept-sine excitations can reveal coexisting solutions.